Planar Selective Leidenfrost Propulsion Without Physically Structured Substrates or Walls

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The Leidenfrost effect allows droplets to be transported on a virtually frictionless layer of vapor above a superheated substrate. The substrates are normally topographically structured using subtractive techniques to produce saw-tooth, herringbone and other patterns and bulk heated, leading to significant challenges in energy consumption and controlled operation. Here, we propose a planar lithographic approach, to levitate and propel droplets using temperature profiles, which can be spatially patterned and controlled in time. We show micro-patterned electrodes can be heated and provide control of the pressure profile and the vapor flow. Using these almost featureless planar substrates we achieve self-directed motion of droplets, with velocities of approximately 30 mms$^{-1}$, without topographically structuring the substrate or introducing physical walls. Our approach has the potential to be integrated into applications, such as digital microfluidics, where frictionless and contactless droplet transport may be advantageous.

The Leidenfrost effect is a case of thin film boiling where, when a droplet contacts a surface that is significantly hotter than the liquid’s boiling point, causing the droplet to levitate on a cushion of its own vapor$^1$. The vapor layer thermally insulates the drop from the hot surface, increasing the droplet’s lifetime, and provides lubrication, which results in increased mobility. Recently, there has been significant interest in this effect to create an effectively perfect superhydrophobic surface, reduce friction and to propel droplets (or a solid block of dry ice undergoing sublimation)$^2$–$^4$. To produce propulsion, asymmetries are introduced into the flow of the exiting vapor, either by structuring the substrate asymmetrically$^5$–$^7$ or by an asymmetric mass distribution$^8$,$^9$. The most common explanation of the propulsion mechanism is that asymmetric vapor flow produces a viscous drag on the levitating component and causes the droplet to translate or rotate in a preferential designed direction$^{10,11}$. Propulsion can be obtained using different structures employing local asymmetries like ratchets$^{12}$–$^{15}$ or global asymmetrical shapes like herringbones$^{6,16,17}$, which are produced using, e.g. subtractive manufacturing techniques. Inducing levitation and controlling drop motion on these surfaces is energy intensive as the entire substrate has to be bulk heated.

In this work, we propose selectively heated substrates to propel and control the motion of droplets using photolithographic patterning of metal films in an asymmetric pattern. The substrates are obtained by planar deposition of electrodes in an asymmetric pattern. By selectively heating the substrate via Joule heating of the electrodes, the input energy to induce levitation is reduced significantly$^{18}$, while patterns of thermal gradients on the substrate are used to rectify the vapor flow for propulsion. We also employ concepts of shaped liquid-vapor interfaces using thermal gradients$^{19}$ to control the position of these highly mobile droplets using negative feedback control. We therefore introduce a highly flexible virtual method to produce the equivalent propulsion and control to ratchet and herringbone structures and physical walls without the need to physically shape the substrate or introduce walls.

As an example of a herringbone-inspired structure we designed resistor line patterns using serpentine electrode designs 40 µm wide, repeated four times, with a 40 µm gap between each line (fig. 1 inset left). This array of 4 serpentine lines acts as a lithographic ‘heater’, which is repeated with a gap of 1.0 mm while patterns of thermal gradients on the substrate are used to rectify the vapor flow for propulsion. We also employ concepts of shaped liquid-vapor interfaces using thermal gradients$^{19}$ to control the position of these highly mobile droplets using negative feedback control. We therefore introduce a highly flexible virtual method to produce the equivalent propulsion and control to ratchet and herringbone structures and physical walls without the need to physically shape the substrate or introduce walls.

![Serpentine Design](image)

FIG. 1. Self-centering herringbone design, showing (inset right) elongation of serpentine pattern when approaching the center of the design and (inset lower left) the standard serpentine design, with line widths of 10 µm, pattern widths of d = 40 µm and a gap between each serpentine line of x = 40 µm, which results in a lithographic ‘virtual pillar’ width of p = 280 µm and a ‘virtual trough’ of 1.0 mm

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pills (here we refer to these as ‘virtual pillars’) and the gaps between these heaters represent the trough region (here we refer to these as ‘virtual troughs’) in a subtractive machined herringbone feature.6

An additional development of the herringbone structure was the addition of elongated (gradient) serpentine patterns to change the current density at different positions relative to the center of the herringbone structure, in order to achieve a self-centering design for droplet motion Figure 1. This results in a temperature distribution with a higher temperature at the edges of the design relative to that in the central region. In the plain herringbone-inspired pattern, with a uniform serpentine design, such a temperature variation will not exist between the sides and the center of the design.

We used a bi-layer lift-off process to produce the metallic pattern on the borosilicate glass substrates. PMGI SF9 is spun onto the substrate at 500 rpm for 10 s, then 6000 rpm for 50 s using a Laurell spinner, resulting in a layer approximately 500 nm thick. This layer is then baked at 200 °C for 5 minutes. Microposit SPR-350 photoresist is spun on top of this baked layer at 700 rpm for 10 s, then 3700 rpm for 50 s using a Laurell spinner, and baked at 110 °C for 3 minutes. A dark field chromium mask is then used in an EVG 620 mask aligner to expose regions of the substrate to UV light for 4 s. The substrate is then developed for 2 minutes using MF-319 developer, ensuring undercutting of the SPR-350 by the SF9. 10 nm of titanium, 20 nm of platinum and 100 nm of gold are then deposited in the same e-beam evaporation process. The excess metal is removed by removing the photoresist below the metal, using Microposit 1165 stripper, leaving the required resistor patterns.

Each wafer contains the desired design and contact pads to ensure an electrical connection. In order to ensure experimental repeatability, a custom rig is designed for the experiments, with an mbed-controlled Peltier cooler with cooling pipes and fan on the underside to maintain a uniform temperature within the stage. Spring loaded electrical contacts are used to pass a current through the patterned electrodes. A USB optical microscope is used to observe the drop motion and a FLIR A40 thermal camera is used to observe the temperature profile of the substrate. An appropriate power is identified for levitation and the array is left for 1 minute for the temperature to equalize before depositing a 20 µl droplet of propan-2-ol (IPA) to observe propulsion.

Figure 2 show the motion of IPA droplets on the herringbone-inspired patterned surfaces with a uniform (Figure 2(a)) and a gradient (Figure 2(b)) serpentine spacing. In Figure 2(a), the droplets travel ‘against the arrows’ in the pattern, which is like the motion of droplets on the features of a herringbone surface.6 It was determined that varying the ‘trough’ width impacted on the ability of the substrate to propel the droplet. Both 0.6 mm and 0.8 mm wide troughs caused droplet propulsion, as seen in the supplementary videos. On the herringbone design, a drop deposited near the center does not remain in the center due to the absence of any restoring force or any physical barriers inhibiting the outward motion. The self-centering design (Figure 2(b)) provides this restoring force due to a negative temperature gradient from the sides towards the center. This negative temperature gradient is caused by the current density gradient due to the elongation of the serpentine shape. The higher temperature regions (near the substrate edges) work as a virtual barrier deflecting the droplets back to the center. Furthermore, this inverts the direction of travel, relative to the non-centering design. In the self-centering design, the droplets travel ‘with the arrows’ due to the thermal gradient. It was also found that a ‘trough’ width for the propulsion of centered droplets was 1.0 mm, as seen in fig. 2. This gap reflects the dimensions of herringbone propulsion described by Soto et al.5

Figure 2(c) shows a typical displacement curve for a droplet on these planar surfaces, where droplets have an initial velocity of 16 mm s⁻¹, and an average velocity of 31 mm s⁻¹ over the first 0.2 s of motion. The droplets do not reach a terminal velocity, and would therefore continue to accelerate on a larger sample. Assuming a viscous drag driven constant propulsion force ($F_p$) and negligible inertial resistance to the motion from the surface pattern, a droplet’s motion can be characterized from the following equation of motion:

$$m\ddot{v}_d = F_p - c_v v_d,$$

(1)
where, $F_p$ is the propulsion force, $m$ is the mass of the droplet, $v_d$ is the droplet velocity and $c_v$ is the coefficient of viscous resistance. The solution to Equation (1) is:

$$v_d = v_t (1 - e^{-t/\tau}), \quad (2)$$

where, $v_t$ is the terminal velocity and $\tau$ is the relaxation time, which is a measure of the drop’s acceleration. The droplet’s position ($x_d$) is given by:

$$x_d = v_t (t + \tau e^{-t/\tau}). \quad (3)$$

Equation (3) agrees well with the experimental data in Figure 2(c). As a result, we obtain the terminal velocity $v_t = 36.4$ mm s$^{-1}$ and the relaxation time $\tau = 0.04$ s.

For the substrate designs with gradient serpentine patterns, multiple designs with different elongation dimensions were tested. It was observed that designs where the central serpentine resistors were elongated by 10% between the edge and the center provided the optimal propulsion performance. A higher elongation resulted in arrays where the central region could not heat sufficiently to support levitation, without compromising outer resistor damage with excessive heat. A smaller elongation did not generate enough of a temperature difference between the edge and the center of the substrate to achieve self-centering.

To qualitatively explain the propulsion observed in our experiments, we develop a 2D model to observe the flow field over a substrate with a sinusoidal temperature distribution: $T_x = T_0 + \alpha \cos(2\pi x/L)$, where $k \geq 3$, as depicted in fig. 3 (a) and (b). This sinusoidal temperature profile is an approximation to the temperature profile on the substrates used in the experiments. Assuming the heat transfer occurs through conduction in the vapor layer and is used for phase change of the liquid, the evaporative velocity $w_0$ is obtained as:

$$w_0 = \frac{\lambda(T_s - T_0)}{\rho v L h}, \quad (4)$$

where, $h$ is the vapor layer thickness, $\lambda$ is the thermal conductivity of the vapor layer, $T_0$ is the boiling point of the liquid, $\rho$ is the density of the vapor layer, $L$ is the latent heat of vaporization of the liquid. Invoking the lubrication approximation, the velocity in the $x$ direction ($u$) and the velocity in the $y$ direction ($v$) can be written as:

$$u = -\frac{1}{2\mu_v} \frac{\partial p}{\partial x} (z-h), \quad (5)$$

$$v = -\frac{1}{2\mu_v} \frac{\partial p}{\partial y} (z-h), \quad (6)$$

where, $p$ is the pressure in the vapor layer and $\mu_v$ is the dynamic viscosity of the vapor. Using eqs. (5) and (6) in the continuity equation ($V \cdot u = 0$) and integrating it from $z = 0$ to $z = h$, with the boundary conditions: at $z = 0, w = 0$ and at $z = h, w = -w_0$, we obtain:

$$\nabla^2 p = c_0 + c_1 \cos \left( \frac{2\pi x}{L} \right), \quad \text{ (7)}$$

where, $c_0 = -12\mu_v \lambda(T_s - T_0)/(\rho v L h^4)$ and $c_1 = -12\mu_v \lambda \alpha/(\rho v L h^4)$ are constants. The solution to eq. (7) is of the form:

$$p(x,y) = \sum_{n=0}^{\infty} \sum_{m=0}^{\infty} P_{m,n} \sin \left( \frac{n\pi x}{L} \right) \sin \left( \frac{m\pi y}{B} \right), \quad \text{ (8)}$$

where, $P_{m,n}$ is a constant obtained as (additional details on the derivation are provided in the supplementary information document):

$$P_{m,n} = \left[ \frac{4c_1}{mn \pi^2} + \frac{4c_1 m}{(m^2 - 4k^2)n \pi^2} \right] \left[ 1 - (-1)^m \right] \left[ 1 - (-1)^n \right] (m\pi/L)^2 \left( 1 - \frac{4k^2}{(m\pi/B)^2} \right). \quad \text{ (9)}$$

In the pressure field obtained from eq. (8), the terms with $c_0$ are the zeroth order solution (Figure 4(a)), which corresponds to the pressure field on a uniformly heated substrate. The coefficient terms with $c_1$ constitute the first order solution, which corresponds to the perturbations due to the temperature field’s
Therefore, the first order velocity field contains information about the vapor flow due to the thermal gradients. Figure 4(b) shows that the vapor flow diverges from regions of higher temperature and converges to regions of lower temperature. In this model, the flow is symmetric about $x$ and $y$, therefore, the net propulsion force from this flow in these directions is zero. However, in a V-shape arrangement, as used in the experiments (Figure 1), this vapor rectification causes a net flow in the $x$ direction, propelling droplets deposited near the centers "against the arrows", similar to that observed in herringbone-like structures obtained through subtractive manufacturing techniques\textsuperscript{6}. Also, as the flow along $y$ diverges from the center of this V-shaped geometry, the drop is unstable and minor perturbations deflect the drop from the center, as observed in Figure 2(a). It is important to note here that this theoretical model is a simplified system compared to the complexities of drop motion in experiments. Approximations such as a sinusoidal temperature field assumption in $x$, a symmetric temperature field in $y$, a flat liquid-vapor interface and a rectangular interface cross-section (as opposed to circular) affect an accurate estimation of the scale of forces acting on the drop. However, the model qualitatively provides information about the direction of vapor flow that drives the motion.

A similar vapor rectification between the electrode patterned regions also occurs in the self-centering design. However, the additional thermal gradient along the electrodes shape the liquid-vapor interface and result in a pressure gradient between the center and the sides. The direction of this pressure gradient generated force can be adjudged by considering a simplified 1D model with a liquid levitating on a substrate with a linear temperature distribution: $T = T_0 + \alpha x$, as depicted in fig. 3 (c). We use this simplified approach to obtain qualitative estimates of the direction of propulsion as opposed to using complete numerical solutions of the liquid-vapor interface profile as used by Sobac et al.\textsuperscript{19}.

Invoking the lubrication approximation, the flow in the $x$-direction can be written by eq. (5). The shear force (per unit width) on the liquid-vapor interface ($F_s$) can be written as:

\[
F_s = \mu_c \int_{-L}^{L} \frac{\partial u}{\partial z} \bigg|_{z=h} \, dx. \quad (10)
\]

The temperature distribution on the substrate creates a difference in pressure between the hot and the cold ends, which shapes the liquid-vapor interface. Considering a first-order approximation, we assume a linearly varying vapor layer thickness: $h = h_0 + \beta x$, as shown in fig. 3 (c). The dynamics of motion induced due to this shaped-liquid interface is similar to the propulsion of uneven solids\textsuperscript{8,14}. With the boundary condition of $p = 0$ at $x = 0$ and $x = L$, using eq. (5) in eq. (10), the shear force can be written as: $F_s = \frac{\beta}{2} \int_{-L}^{L} p \, dx$. The total propulsion force ($F_p$) is given by the sum of the shear force ($F_s$) and the normal force ($F_n = -\int_{-L}^{L} p \, dx \sin \beta$) on the liquid-vapor interface in the $x$ direction:\textsuperscript{8}

\[
F_p \approx \frac{\beta}{2} \int_{-L}^{L} p \, dx \quad (11)
\]

The normal force due to the pressure field will balance the weight of the levitating liquid ($mg$):

\[
\int_{-L}^{L} p \, dx \cos \beta = mg. \quad (12)
\]

Also, the net torque on the levitating liquid will be zero:

\[
\int_{-L}^{L} px \, dx \cos \beta = \frac{mgH \sin \beta}{2}. \quad (13)
\]

The above two equations provide the pressure field $p$ and $\beta$ (derivation details in supplementary information). The normal pressure due to the deformed liquid-vapor interface contributes to the propulsion\textsuperscript{8}, where the direction of motion is from the region of higher temperatures to lower temperatures, in agreement with the numerical predictions of Sobac et al.\textsuperscript{19}. The self-centering designs in our experiments rely on this force to center droplets. As a drop traverses off-center, the negative temperature gradient towards the center shapes the liquid-vapor interface, which produces a force towards the
center. Once the droplet reaches the center, the vapor rectification between the electrodes propels it in the x direction.

In this work, we have used planar surfaces to propel droplets. We have used lithographic techniques to pattern micro-heater arrays on a substrate, which provide selective heating to sustain levitation. The asymmetric macro-pattern of these arrays has been tailored to provide a temperature differential which propels and, using a negative feedback control, self-centers droplets on the substrate, similar to substrates using physical features produced by subtractive machining. Our concepts of motion induced as a result of a thermal gradient using selective programmable heating can be used to develop energy efficient devices for droplet motion control in microfluidic systems, which can be fabricated more readily using standard lithographic techniques.

SUPPLEMENTARY MATERIAL

Please see the supplementary material for additional information on the analytical model, as well as a video of propulsion in a non-centered design (Figure S1) and a centered design (Figure S2).

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